

RESERVE COPY

PATENT SPECIFICATION

615,015



Convention Dates
(United States of America)

Oct. 25, 1945.
April 9, 1946.

Corresponding Applications
in United Kingdom

No. 24141/46 }
No. 24142/46 } dated Aug. 14, 1946.

(One Complete Specification left under Section 91 (2) of the Patents and Designs Acts, 1907 to 1946.)

Specification Accepted: Dec. 31, 1948.

Index at acceptance:—Class 85, A1a.

COMPLETE SPECIFICATION

Improvements in or relating to Devices for Actuating Well
Pipe Slips

We, BYRON JACKSON Co., a Corporation organized under the laws of the State of Delaware, United States of America, of 2301, East Vernon Avenue, City of Vernon, State of California, United States of America, do hereby declare the nature of this invention and in what manner the same is to be performed, to be particularly described and ascertained in and by the following statement:—

10 This invention relates to power-operated actuating means for slips such as are used for supporting an oil well drill pipe and casing.

15 In accordance with the present invention, the improved device for actuating well pipe slips includes a substantially horizontally extending slip-supporting member supporting pipe slips adjacent
20 one of its ends, means connected to said member adjacent its other end for bodily reciprocating said member in a vertical direction, means pivotally connecting said member to said reciprocating means
25 for vertical swinging movement relative thereto, and means for releasably locking said member against pivotal movement relative to said reciprocating means.

30 The present invention also provides a fluid-operated device for actuating well pipe slips, including a slip-supporting arm carrying pipe slips adjacent one end thereof and means connected to said arm adjacent the other end thereof for bodily
35 moving said arm upwardly and downwardly, said last-named means comprising telescopically arranged cylinder and plunger elements, one connected in supporting relation to said arm and the other
40 being mounted in a stationary position, means closing opposite ends of said plunger element, a piston in said plunger element and dividing the interior thereof into two compartments, a piston rod

secured at one end thereof to said cylinder 45 element and at its other end to said piston for holding the piston stationary with respect to the cylinder element, and passage means for admitting fluid under pressure selectively to either of said com- 50 partments.

In a preferred embodiment of the invention the improved device for actuating well pipe slips includes a slip-supporting arm supported adjacent one end 55 thereof on vertically reciprocable actuating means for reciprocation therewith, the other end of said arm being shaped to provide a supporting ring defining a circular opening, means on said support- 60 ing ring rotatably supporting a slip carrier ring in said opening and pipe slips suspended from said carrier ring, said supporting ring and said carrier ring each having a segment thereof detachable from 65 the remainder of the ring, at least at one end of the segment, to provide a lateral pipe opening whereby said rings may be applied laterally to a pipe intermediate its ends. 70

In the process of drilling and completing oil wells it is customary to support the drill pipe in the rotary table while making up the threaded joints between the pipe sections. The pipe is supported by tapered 75 wedge slips seated in a tapered opening in the master bushing supported in the seried and removed manually. Because of rotary table, the slips usually being in the weight of the slips, the services of two 80 men are required for this operation, which is not only arduous but also hazardous. When drilling below ten thousand feet, which is now common, the number of sections or stands into which the drill 85 string is broken down when it is removed from the well, is well in excess of one hundred. The slips must be inserted and

BEST AVAILABLE COPY

removed each time a section is broken out or made up, and this repeated handling of the slips induces fatigue in the men handling the slips, which not only slows
6 down the operation but also renders them less alert to danger.

Various means have heretofore been proposed for moving the slips to and from operating position, but such means have not proved to be practical for various
10 reasons, such as insufficient range of movement of the slips to accommodate a wide range of pipe sizes, lack of dependability under all operating conditions, cumbersomeness of the equipment, interference
15 with other operations, such as tonging the joints, and other practical drawbacks.

One of the principal objects of this invention is to provide power-operated slip actuating means which is simple and
20 dependable, which may be disposed in an out-of-the-way position when not in use, and which may be quickly and easily moved into operating position.

It is a further object of this invention to provide a device of the aforesaid type which may be used with a wide range of
25 sizes of pipe slips, to support a pipe of a size up to the maximum which can be supported in a standard master bushing.

A still further object of this invention is to provide a pipe slip operating mechanism which is dependent of the construction of the rotary table and hence is
30 adaptable to use with any make or size of table.

It occasionally happens that the derrick crown block is not vertically aligned with the opening in the rotary table, thus causing
40 the suspended drill string to hang slightly off-center with respect to the table opening. In this event, when setting the slips the pipe must be forced laterally into central position.

A still further object of this invention is to provide a pipe slip operating mechanism which is positively acting in both
45 slip-releasing and slip-setting directions. By providing means for forcing the slips downwardly toward their seats, the pipe is centered in the table opening without the necessity of relying on the drag of the pipe on the slips to pull the latter
50 downwardly and centre the pipe in that way. The exertion of a positive downward force on the slips also enhances their gripping action, especially when only a short length of pipe is being supported.

A still further object is to provide an automatic safety latch which is normally
60 inoperative but which becomes operative to prevent premature setting of the slips in the event of the supply of actuating fluid being inadvertently cut off or its pressure dropping below a predetermined
65

value while the slips are in raised position.

A still further object is to provide a power-operated slip actuating device which incorporates a safety releasing connection between elements thereof, which
70 prevents damage to the mechanism in the event of the pipe elevator being lowered too far and engaging the slip supporting element, and, also, in the event of the
75 drill string being inadvertently raised by the hoisting mechanism while the slip-actuating mechanism is locked in slip-setting position.

A still further object is to provide a power-operated slip actuating device in which these slips are supported by a continuous annular ring which in turn is rotatably supported by a second continuous ring, and in which both rings
80 include a pivoted or detachable segment to permit their being applied to and removed laterally from the pipe. In this manner, the device may be moved into operating
85 position around the pipe or removed therefrom while the pipe is suspended by the hoisting mechanism. Otherwise it would be necessary to temporarily support the pipe in the rotary table by other slips and break a joint and then
90 lower the rings over the upper end of the pipe.

A still further object is to provide means for temporarily securing the segments of the slip carrier ring to the corresponding
100 sections of the supporting ring when the rings are opened up to enable them to be applied to or removed laterally from the pipe.

A still further object of this invention is to provide a simple and effective means
105 whereby the slips may be quickly and easily detached from their supporting ring, thus facilitating the substitution of different sized slips when running a pipe
110 string made up of different sizes of pipe, subs and drill collars.

Other objects and advantages of the invention will be apparent from the following detailed description of one
115 embodiment of the invention, reference being made to the accompanying drawings wherein:

Fig. 1 is a view in side elevation of the device in operating position with
120 parts cut away to show the slips in set position; and

Fig. 2 is a top plan view of the device of Fig. 1; and

Fig. 3 is a vertical sectional view taken
125 on the line III—III of Fig. 2; and

Fig. 4 is a sectional view taken on the line IV—IV of Figs. 1 and 3; and

Fig. 5 is a sectional view taken on the line V—V of Fig. 2;
130

615,015

3

Fig. 6 is a view in side elevation of a modified cylinder assembly incorporating an automatic safety latch;

Fig. 7 is a horizontal sectional view taken on line VII—VII of Fig. 6;

Fig. 8 is a vertical sectional view taken on line VIII—VIII of Fig. 7;

Fig. 9 is a top plan view of the slip-supporting arm and carrier ring in open position, with the carrier ring segments temporarily secured to the arm and gate;

Fig. 10 is a view taken on line X—X of Fig. 9; and

Fig. 11 is a vertical sectional view taken on line XI—XI of Fig. 9.

Referring to the drawings, reference numeral 1 denotes a rotary table supported on the usual X-beams 2 forming part of the sub-base of the derrick structure. A cylinder assembly, generally designated 3, is rigidly supported in predetermined relation to the rotary table in any suitable manner. In the present instance it is clamped to the X-beams 2.

Referring particularly to Figs. 1 and 3, it will be observed that the cylinder assembly 3 comprises generally a power cylinder 4 and a guide cylinder 5, rigidly connected together by vertically spaced plates 6 and 7. The lower plate 7 is supported on a pair of channel beams 8, 8 bridging the space between the X-beams 2 and supported thereon. The cylinder assembly is clamped to the X-beams by clamp bolts 9, herein four in number, extending between the plate 7 and a pair of transversely extending sections of pipe 10 engaging the under side of the X-beams 2. It will be understood that the derrick floor is built up around the rotary table and the cylinder assembly, but this has been omitted from the drawing in order to illustrate more clearly the cylinder assembly mounting.

The power cylinder 4 is of the double acting plunger type and includes a plunger 12 projecting through the upper end of the cylinder and slidable therein. The upper end of the plunger is closed by a head 13 to which is attached a swivel base 14, to be hereinafter described. The lower end of the plunger is closed by a head 15 threadably connected thereto. Mounted concentrically within the cylinder and plunger is a tubular member 16 within which extends a second tubular member 17 spaced inwardly therefrom to provide an annular fluid passage 18 between the members. The inner tubular member 17 also provides a central fluid passage 19. The outer member 16 is rigidly secured at its lower end to a cylinder head 20 and is thus held stationary with respect to the cylinder.

The inner member 17 is secured at its

lower end to the outer member 16, as by welding at 21 which also serves to close the lower end of the annular fluid passage 18. The members 16 and 17 extend through the lower plunger head 15, which is provided with suitable packing 22 forming a fluid-tight sliding joint with the outer wall of the member 16. A piston 23 is secured to the upper extremities of the members 16 and 17 and engages the inner wall of the plunger 12 to divide the interior thereof into upper and lower compartments 24 and 25 into which actuating fluid is alternately admitted to move the plunger upwardly or downwardly.

It will be observed that the inner tubular member 17 extends through the piston 23, thus communicating the fluid passage 19 with the compartment 24. One or more lateral ports 26 are provided in the wall of the outer tubular member 16 immediately below the piston 23 thus communicating the annular fluid passage 18 with the compartment 25. A conduit 27 is connected to the lower extremity of the inner tubular member, and a conduit 28 is connected to the cylinder head 20 and communicates with the annular passage 18. The conduits 27 and 28 are connected at their other ends to a four-way reversing valve, not shown but preferably located adjacent the driller's position, whereby the driller may selectively admit pressure fluid from a suitable source to either the upper compartment 24 or the lower compartment 25 of the plunger 12, and thus raise or lower the plunger. The reversing valve is, of course, arranged so that when pressure fluid is admitted to one of the compartments the other compartment is exhausted.

The guide cylinder 5 slidably receives a sleeve 30 to the upper extremity of which is secured a head 31 similar to the head 13 and also attached to the swivel base 14. The guide cylinder and sleeve serve merely to relieve the plunger 12 and its cylinder 4 of lateral stresses which would be otherwise imposed thereon by the overhanging weight of the slip assembly (Fig. 1). The sleeve 30 extends downwardly a substantial distance below the lower end of the cylinder 5 when in its lowermost position, so that when it is raised to its uppermost position it will extend substantially throughout the entire length of the cylinder 5 and thus obtain the maximum guiding effect. The downwardly projecting lower end of the sleeve 30 is enclosed within a housing 32 depending from the cylinder 5. The housing may, if desired, be filled with lubricant into which the

sleeve will dip each time it is lowered. It is considered preferable, but not essential, that bearing bushings of bronze or the like be provided in the guide cylinder 5, as at 33 and 34. A similar bushing 35 may be provided also in the power cylinder 4.

Adjacent the inner end of the swivel base 14 a swivel plate 36 is mounted thereon for horizontal swiveling about the axis of an upright pivot bolt 37. The swivel plate is provided with a pair of upstanding ears 38, 38 which receive a transverse pivot pin 39 for pivotally supporting a slip-supporting arm 40 extending inwardly over the rotary table. As shown in Fig. 4, the arm 40 is of inverted U-shape in cross-section, and is normally latched against both horizontal swiveling and vertical pivotal movement by a latch 41 pivotally mounted on the swivel base 14. As shown in Figs. 3 and 4, the latch is pivotally mounted on a transverse pin 42 journaled in ears 43, 43 formed on the swivel base 14, and is keyed to the pin 42 as by tapered pins 44, 44. A torsion spring 45 normally urges the latch into its upright position wherein it engages a transverse latch pin 46 extending between the side walls of the arm 40. The latch pin 46 is purposely made relatively small and of insufficient strength to withstand a severe strain or blow, so that in the event the arm 40 is inadvertently subjected to an excessive strain in either upward or downward direction, the latch pin will shear and allow the arm to pivot upwardly or downwardly relative to the swivel base. The provision of the swivel plate 36 and latch 41 renders it possible to move the arm, and the slip assembly supported thereby, into an inoperative position clear of the table when not in use, such as during drilling. The latch pivot pin 42 is provided with an integral handle portion 47 at one end thereof, to permit manual manipulation of the latch out of engagement with the latch pin 46. With the latch released, eyes 48, 48 on the arm (Fig. 2) may be engaged by the usual catline and the arm raised upwardly about the pivot pin 39 until it clears the upper end of the latch 41, whereupon the arm and swivel plate 36 may be rotated about the upright swivel bolt 37 into any desired inoperative position. When returning the arm to its operative position it is unnecessary to manually retract the latch, in view of the provision of a stop 49 on the latch which, in conjunction with the torsion spring 45, normally retains the latch in a position wherein its beveled nose portion 50 is engaged by the latch pin 46 as the arm descends, thus

camming the latch open. After riding downwardly along the beveled face of the latch, the latch pin engages a projecting ledge 51 on the latch, thus limiting the downward movement of the arm and allowing the latch pin to enter the slot 52 in the latch as the latch returns to upright position under the influence of the spring 45. Although the latch has been referred to above as provided with only a single nose portion 50, ledge 51 and slot 52, it will be seen from Fig. 4 that these parts are duplicated at opposite sides of the latch, the slotted portion having close clearance with a pair of internal ribs 53, 53 formed on the inner wall of the arm 40. In this manner the latch pin is subjected to pure shear stress at the two planes at the opposite sides of the latch, rather than to bending stress. An excessive strain will therefore shear the pin cleanly, making it easy to replace it.

Referring again to Figs. 1, 2 and 5, it will be observed that the arm 40 is forked at its inner end to form a semi-circular ring 55. A cooperating semi-circular ring 56 is pivotally connected at 57 to the ring 55 and is releasably connected thereto at a diametrically opposite point, as by a retractable pin 58. The rings 55 and 56 cooperate to provide a circular recess in which is rotatably supported a slip-carrier ring generally designated 59. As shown most clearly in Figs. 2 and 5, rollers 60 are mounted at spaced intervals on the rings 55 and 56 and engage a recess 61 in the outer periphery of the carrier ring 59 for rotatably supporting the latter and confining it against either upward or downward movement relative to the rings 55 and 56. Referring particularly to Fig. 5, it will be observed that the recess 61 is defined by upper and lower frusto-conical trackways 62 and 63 which are selectively engaged by the rollers 60. Each roller is journaled on a pin 64 secured in the ring 55 or 56, by a roller bearing 65 which may be lubricated at intervals through a fitting 66.

The slip-carrier ring 59 comprises two segments 67 and 68, detachably connected together at 69 and 70. The ring segment 67 embraces an arc of approximately 240° and supports two slips of a three-slip assembly, while the segment 68 supports the third slip. The internal diameter of the carrier ring is at least as large as the opening through the master bushing 71 in the rotary table, and the slips are suspended from the carrier ring in such a manner that as the ring is lifted the slips move radially outwardly into positions wherein they are disposed

615,015

5

entirely outside the opening through the ring and the master bushing. It is, therefore, impossible for any projections on the drill pipe, such as tool joints and casing protectors, to drag on the slips as the pipe is raised or lowered.

As shown in Fig. 2, three pairs of pivot lugs 75, 76, 76 and 77, 77 project downwardly from the lower surface of the carrier ring segments 67 and 68, each pair of lugs forming a pivotal supporting connection for a slip rein 78. A slip 79 is pivotally connected to the lower end of each rein on a horizontal pivot at 80 (Fig. 1). The slips may be of conventional design, except for the pivotal connection to the reins, and hence further detailed description thereof is deemed unnecessary. The pivotal connection between the upper ends of the reins and the carrier ring is of the quick-detachable type, and consists of a short pivot pin 81 (Fig. 5) projecting outwardly from each side member of the rein and integral therewith. The pins 81 are flattened on two opposite sides for insertion into the bores in the pivot lugs 75 through slots 82 in the latter, when the reins are swung into a predetermined position. When it is desired to replace the slips 79 with slips of a different size, the reins are swung in a clockwise direction, as viewed in Fig. 5, approximately 120° from the position shown in that figure, thus aligning the flattened sides of the pivot pins 81 with the slots 82 in the pivot lugs 75. The reins may then be detached from the carrier ring by moving the reins bodily inwardly toward the axis of the ring and removing the pivot pins from the lugs through the slots 82. Another set of slips, with substantially identical reins 79 attached thereto may then be attached to the ring by reversing the foregoing steps. The foregoing arrangement eliminates the use of detachable parts such as separate bolts and nuts or the like which might work loose while the device is in use, and it also possesses the advantage of avoiding loss of time in removing separate pivot bolts and their securing means.

In order to enable the slips to be retracted laterally a sufficient distance to clear any size pipe or projection thereon which can pass through a standard master bushing, the slips must be raised completely above the top of the master bushing. A standard master bushing is provided with a flat upper surface, and hence it is necessary to provide means for guiding the lower ends of the slips inwardly during the initial lowering movement, in order to prevent their hanging up on the flat upper surface of

the master bushing. For this purpose a two-piece guide plate 85 (Fig. 1) is positioned on the master bushing and properly centered by the Kelly bushing square in the master bushing. The upper surface of the guide plate is tapered downwardly and inwardly at a small angle of the order of 30°, as indicated at 86, to provide a sloping surface down which the lower ends of the slips will ride as the carrier ring is initially lowered. An adjustable stop is provided for adjustably limiting the upward movement of the base 14, in this instance a rod 88 being secured to the base 14 and extending downwardly through aligned openings in the plates 6 and 7, and having an adjustable stop nut 89 on its lower end. The opening 90 in the lower plate 70 is large enough to permit the nut 89 to pass through it and engage the plate 6 to limit upward travel of the base 14, arm 40 and slips.

The mode of operation of the device will be apparent from the foregoing description. When not in use, such as while drilling, the arm 40, with the carrier ring 59 and slips supported thereby, extends in the opposite direction from that shown in Figs. 1 and 2. When the drilling bit becomes dull and it becomes necessary to withdraw the drill string from the wall to replace the worn bit, the drill string is raised until the first joint of drill pipe is above the rotary table. The power cylinder 4 is first actuated to raise the base 14 and the arm 40 to their uppermost position. The catline is then connected to the arm 40 by the eyes 48 and the arm is pivoted upwardly about the transverse pivot 39. With the pivoted sections 56 and 68 of the arm 40 and the carrier ring swung into open position, the arm is swung horizontally about the upright pivot 37 (Fig. 3) into operating position over the rotary table. The arm is then lowered to a horizontal position, in which position it is latched by the latch 31. The pivoted ring segments 56 and 68 are then closed and locked by their respective pins 58, 69 and 70. The device then assumes the position indicated in dotted lines in Fig. 1, with the lower ends of the slips engaging the sloping surface 86 on the guide plate 85. Upon actuation of the reversing valve by the driller, pressure fluid is admitted through the conduit 28, annular passage 18 and port 26 to the lower plunger compartment 25, thus positively forcing the plunger 12 downwardly and setting the slips in the tapered bowl in the master bushing. The compartment 24 is simultaneously exhausted through the central fluid

- 6

- 615,015

passage 19 and the conduit 27.

After breaking the joint between the Kelly and the stand of pipe gripped by the slips, and setting the Kelly aside, the elevator is latched about the upper end of the pipe and the drill string is raised. With a long string of pipe supported by the slips, the weight of the pipe will hold the slips set even though the reversing valve be actuated to lifting position, hence this may be done before the pipe is raised. In that event, raising the pipe by the hoisting mechanism allows the hydraulic actuator to automatically raise the arm 40 and the slips as soon as the weight of the pipe is removed from the slips. Upward movement of the plunger 12 is effected by admitting pressure fluid through conduit 27 and passage 19 to the upper plunger compartment 24, the lower compartment 25 being exhausted through passage 18 and conduit 28.

It should be noted that the pivotal connection 80 between the slips and the reins 78 is located above and offset slightly inwardly of the center of mass of the slips, so that as the slips are raised out of the master bushing and move upwardly and outwardly along the tapered surface 86 of the guide plate 85, they are maintained in upright position, as indicated in dotted lines in Fig. 1. It will further be noted with reference to Fig. 5 that cooperating abutments 88¹ and 89¹ are provided on the reins 78 and the carrier ring 59, respectively, which engage each other when the slips are moved inwardly into engagement with the pipe. The downward force exerted by the power cylinder and plunger is thus transmitted from the carrier ring to the reins through these abutments 88¹ and 89¹ when the slips are set, thereby relieving the strain on the pivot lugs 73 and pivot pins 81.

It is customary, when breaking the joints between drill pipe sections, to first loosen the joint by the use of tongs and then complete the unscrewing of the joint by rotating the table and the drill string supported thereby while holding the upper stand of pipe against rotation. It is because of this practice that it is necessary to rotatably mount the carrier ring 59 in the supporting ring 55, 56. The rollers 60 enable the carrier ring to rotate freely with the pipe and slips while the arm 40 and supporting ring exert downward pressure on the carrier ring and slips. The anti friction mounting also facilitates manual rotation of the carrier ring into proper position for opening the pivoted section 68 prior to moving the device into inoperative position.

In Figs. 6, 7 and 8 there is shown a slightly modified form of the cylinder assembly, which has incorporated therein an automatic safety latch, generally designated 101. Referring particularly to Figs. 7 and 8, a body member 102 is mounted on the upper surface of the upper plate 6 extending between the cylinders 4 and 5. The interior of the body member provides a rectangular recess in which a latch member 103 is mounted for limited sliding movement in a horizontal plane. The latch member is constantly urged to the left, as viewed in Figs. 6 and 7, by compression springs 104 and 105 mounted in guide recesses in the body member 102, and is forced in the opposite direction by fluid pressure.

A pair of parallel cylinder bores 106 and 107 are formed in the left-hand side of the body member, in which a pair of plungers 108 and 109 are slidable. The plungers are secured to the adjacent side of the latch member 103 by cap screws 110 and 111, and are sealed to the cylinder bores by cup washers 112 and 113. The cylinders 106 and 107 communicate respectively with the fluid supply conduits 28 and 27 (Fig. 6) through conduits 114 and 115. Inasmuch as one or the other of the conduits 27 and 28 is connected to the source of pressure fluid to either raise or lower the slip mechanism, it will be apparent that the latch member will be retained in its right-hand position shown in Figs. 7 and 8 as long as pressure fluid is supplied to the actuating cylinder at a pressure high enough to overcome the combined forces of the springs 104, 105. The spring force may be varied by suitable adjustment of the abutment screws 116, 117 against which the springs seat. These screws have threaded engagement with the body member 102 to enclose the top adjusted position by lock nuts 118, 119.

A housing member 120 is fitted over the body member 102 to enclose the top and sides of the latch member 103. Laterally extending flanges 121, are formed on the housing member to provide for securing it and also the body member 102 to the plate 6, as by cap screws 122. As shown in Fig. 8, the body member 102 and housing member 120 are provided with interlocking shoulders at 123 and 124 whereby the body member is constrained against movement relative to the housing member in the direction of movement of the latch member 103.

It will be observed by reference to Fig. 8 that the housing member 120, body member 102 and plate 6 are provided with vertically aligned bores 125, 126 and 127, respectively, through which the rod 88, 130

615,015

7

extends. In this form of the invention a sleeve 128 is fitted over the rod 88 and is connected thereto for axial adjustment on the rod. As shown, the upper end of the sleeve is secured, as by welding, to a nut 129 threaded on the rod. A lock nut 130 secures the sleeve in adjusted position. The sleeve 128 terminates at its lower end a short distance above the stop nut 89 on the lower end of the rod 88, thus providing a downwardly facing shoulder 131 which is adapted to engage the upper surface of the latch member 103 when the latter is in latching position.

Referring to Fig. 7, it will be noted that the latch member 103 is provided with a relatively large circular opening 132 extending vertically therethrough, which is preferably of the same diameter as the bores 125 and 126 and is disposed concentric with the latter when the latch member is in its released position, as shown in the drawing. This opening provides clearance for unobstructed passage of the sleeve 128 therethrough. A smaller opening 133 is also provided in the latch member, offset from the opening 132. The opening 133 is of approximately the diameter of the rod 88 but smaller than the sleeve 128, and hence when the latch member is moved to the left to dispose the opening 133 concentric with the rod 88, the shoulder 131 on the lower end of the sleeve 128 will overlie the latch member and will be prevented from passing therethrough. It will be understood, of course, that the rod and sleeve must be raised to dispose the shoulder 131 above the latch member before the latter can be moved into latching position. When the slip mechanism is in raised position, as determined by engagement of the stop nut 89 with the under surface of the plate 6, the shoulder 131 is at approximately the position indicated by dotted line at 135, Fig. 8. Ample clearance is thus provided between the raised position of the shoulder and the upper surface of the latch member, in order to afford sufficient time for the latch member to move into latching position before the shoulder 131 descends to latching position.

It will be apparent that by subjecting the latch member, through the intermediary of one or the other of the plungers 108, 109, to fluid pressure from either supply line 27, or 28, the latch member will be retained in its inactive position as long as the supply of pressure fluid is maintained at or near the desired working pressure. The pressure fluid is admitted alternately to the supply lines 27 and 28 under the control of a four-way valve, so that as the pressure is cut

off from one line it is simultaneously admitted to the other line. If, however, while the slips are in raised position the supply line should break, or the pressure should drop due to break-down of the pump or compressor or for any other reason, the springs 116, 117 will move the latch member to the left to dispose the small opening 133 therein into a position concentric with the rod 88. The descent of the slip mechanism will thus be interrupted by engagement of the shoulder 131 on the sleeve 128 with the upper surface of the latch member.

In this embodiment of the invention we have also shown resilient bumpers in the form of discs 138 and 139 of rubber, Neoprene or the like, loosely surrounding the rod 88 above and below the plate 6 and safety latch 101. When the slips are raised, their upward travel is quite rapid at the end of the stroke, and the metal-to-metal impact of the stop nut 89 with the under side of the plate 6 would be severe were it not for the cushioning effect of the resilient disc 139 interposed between those elements. When lowering the slips, after the slips have seated in the master bushing bowl the downward force exerted by the pressure fluid causes the arm 40, swivel base 14 and associated parts to tend to descend further, exerting a strain on the arm 40. This condition is alleviated by providing the resilient disc 138 above the safety latch and by adjusting the nut 129 on the rod 88 so that the nut engages the disc at the same time that the slips assume their set position. The resilience of the disc opposes the downward force exerted by the pressure fluid and thus relieves the arm 40 and related parts of unnecessary strain.

Referring to Figs. 9 and 11, means are provided for temporarily securing the segment 67 of the slip-carrier ring 59 to the corresponding section 55 of the supporting ring when the rings are opened up to apply the device to the pipe or remove it therefrom. An inverted U-shaped locking member 140 has one leg 141 thereof permanently mounted in a bore 142 in the arm 40, the other leg 143 being adapted to be inserted either in a bore 144 in the ring segment 67 or in a bore 145 in the arm 40 when the locking member is not in use. The leg 141 extends downwardly below the arm 40 and is provided with a washer at its lower extremity to limit its upward retraction to that necessary to shift the leg 143 from the bore 144 to the bore 145. When the ring segment 67 is thus locked to the ring section 55 and arm 40, the pivot lugs thereon are in position to be aligned with

the corresponding pivot lugs on the mating ring segment 68 when the pivoted ring section 56 is swung to closed position.

Inasmuch as the slip-carrier ring segment 68 is completely detached from its mating segment 67 when the supporting ring section 56 is swung to open position, and is supported only by the single roller carried by the ring section 56, it is necessary to provide other means for temporarily securing it to the ring section 56. A simple and inexpensive means for this purpose is shown in Figs. 9 and 10, and comprises a U-shaped clamp 146 adapted to embrace the ring section 56 and ring segment 68 and to be secured to the latter by a pin 147 received in a bore in the segment 68. The clamp comprises a single continuous rod bent to provide a pair of diverging upper arm portions 148 and 149 forming an eye at their juncture for the reception of the pin 147, their outer ends being bent downwardly to form vertical legs 150 extending downwardly along the outer surface of the ring section 56, and then bent inwardly to provide lower horizontal arms 151 and 152 underlying the ring section 56 and ring segment 68 to support the latter. A pair of hand grips 153, 154 may be secured by welding to the legs 150 to facilitate handling the clamp.

It will be understood that while the device is in use the locking member 140 is in its inactive position with the leg 149 inserted in the bore 145, and the clamp 146 is detached from the ring sections 56 and 68. When the device is to be removed from around the string of pipe the locking member is shifted to its locking position in engagement with the bore 144 in the ring segment 67, and the clamp 146 is applied to the ring section 56 and ring segment 68 and locked to the segment 68 by the pin 147. The retaining pins 69 and 70, holding the segments 67 and 68 together, and the latch pin 58 are then removed and the ring section 56 is swung outwardly about its pivot pin 57, carrying the ring segment 69 with it. After releasing the latch 41, the arm 40 and the rings and slips carried thereby may then be pivoted upwardly about the arm pivot 39 by connecting the catline to the eyes 43 (Fig. 2), and when so raised the parts may be swung horizontally about the vertical swivel pin 37 (Fig. 3) into an inactive position alongside the rotary table.

It will be apparent from the foregoing description of one embodiment of the invention and a modification thereof that it possesses the practical advantages of simplicity, ruggedness and dependability under the severe operating conditions to

which a device of this type is subjected. The mounting of the device independently of the rotary table renders it universally adaptable to all makes and sizes of rotary tables. By providing an opening in the carrier ring 59 at least as large as the opening through the master bushing, and by providing for sufficient set-back of the slips to clear those openings, it is possible to handle pipes of any size up to the maximum size which can be run through and supported in a standard master bushing. The quick-detachable connection between the carrier ring 59 and the slip reins 73 reduces to a minimum the time required to change to different sized slips. The mounting of the arm 40 for vertical pivotal movement and horizontal swiveling with respect to the base 14 provides a simple arrangement for moving the arm and the parts supported thereby into an inoperative position wholly clear of the rotary table. The power cylinder and plunger are equally adaptable to operation by compressed air, liquid or steam. Other advantages inherent in the device will be apparent to those familiar with deep well drilling operation.

Although there has been described a specific embodiment of the invention, it is to be understood that various modifications may be made therein without departing from the spirit of the invention or the scope of the appended claims.

Having now particularly described and ascertained the nature of my said invention, and in what manner the same is to be performed, we declare that what we claim is:—

1. A device for actuating well pipe slips, including a substantially horizontal extending slip-supporting member supporting pipe slips adjacent one of its ends, means connected to said member adjacent its other end for bodily reciprocating said member in a vertical direction; means pivotally connecting said member to said reciprocating means for vertical swinging movement relative thereto, and means for releasably locking said member against pivotal movement relative to said reciprocating means.

2. A device according to claim 1, 120 wherein said means for bodily reciprocating said slip-supporting member is itself vertically reciprocable.

3. A device according to claim 1 or 2, wherein said member is mounted on said reciprocating means for horizontal swinging movement relative thereto, said locking means being effective releasably to lock said member both against vertical pivotal movement and against horizontal 180

615,015

9

swinging movement relative to said reciprocable means.

4. A device according to claim 1, 2 or 3, including a fluid-actuated cylinder and piston assembly adapted to be mounted in an upright position adjacent a slip-supporting bowl, said slip-supporting member being connected adjacent one end thereof to the movable element of said assembly, extending laterally therefrom and having its other, slip-carrying end normally disposed above said bowl, said member being pivotally connected to said movable element for vertical pivotal movement relative thereto, said locking means being adapted releasably to secure said member rigidly to said moveable element, whereby reciprocation of said moveable element moves said slips into or out of engagement with said bowl.

5. A device according to any of the preceding claims, wherein said locking means includes a safety element adapted to fail and release said locking means when said member is subjected to a predetermined load greater than normal but less than that which would damage other elements of said device.

6. A device according to claim 5, wherein said locking means comprises a latch member and a cooperating latch pin, one mounted on said slip-supporting member and the other mounted on said reciprocating means or movable element, said latch pin being adapted to fail and thereby permit pivotal movement of said slip-supporting member relative to said reciprocating means when said slip-supporting member is subjected to said predetermined load.

7. A device according to claim 3, wherein said locking means comprises a latch member and a cooperating latch element, one mounted on said slip-supporting member and the other mounted on said reciprocating means, said latch member and latch element cooperating to prevent vertical pivotal movement of said slip-supporting member relative to said reciprocating means, and said latch member having surfaces thereon engageable with opposed surfaces on the member on which said latch element is mounted, to prevent horizontal swinging movement of said slip-supporting member relative to said reciprocating means.

8. A device according to any of the preceding claims, including a base member adapted to be raised and lowered by power-actuating means, an element mounted on said base member for rotation relative thereto about a substantially vertical axis, an arm member carrying the pipe slips adjacent one end and pivotally

connected adjacent its other end to said element for swinging movement relative to the latter about a substantially horizontal axis, said locking means being effective releasably to latch said arm member in a substantially horizontal position and to lock said element against rotation relative to said base member.

9. A fluid-operated device for actuating well pipe slips, including a slip-supporting arm carrying pipe slips adjacent one end thereof and means connected to said arm adjacent the other end thereof for bodily moving said arm upwardly and downwardly, said last-named means comprising telescopically arranged cylinder and plunger elements, one connected in supporting relation to said arm and the other being mounted in a stationary position, means closing opposite ends of said plunger element, a piston in said plunger element and dividing the interior thereof into two compartments, a piston rod secured at one end thereof to said cylinder element and at its other end to said piston for holding the piston stationary with respect to the cylinder element, and passage means for admitting fluid under pressure selectively to either of said compartments.

10. A device according to claim 9, wherein said fluid passage means extends through said piston rod.

11. A device according to claim 9 or 10, wherein said piston rod comprises concentric tubular conduit members providing a central fluid passage in the inner conduit member and an annular fluid passage between the members, said central passage communicating with one of said compartments and said annular fluid passage communicating with the other compartment, and means is provided for admitting pressure fluid selectively to either of said fluid passages.

12. A fluid-operated device for actuating well pipe slips, including a slip-supporting arm carrying pipe slips adjacent one end thereof and means connected to said arm adjacent the other end thereof for bodily moving said arm upwardly and downwardly, said means comprising a vertically disposed power cylinder and a guide cylinder mounted in parallel relation thereto, a plunger reciprocable in said power cylinder and a guide sleeve reciprocable in said guide cylinder, a base member secured to the free ends of said plunger and guide sleeve, means mounting said slip-supporting arm on said base member, and means for admitting pressure fluid to said power cylinder to reciprocate said plunger, guide sleeve, base member and arm as a unit.

10

615,015

13. A device for actuating well pipe slips, including a slip-supporting arm supported adjacent one end thereof on vertically reciprocable actuating means for reciprocation therewith, the other end of said arm being shaped to provide a supporting ring defining a circular opening, means on said supporting ring rotatably supporting a slip carrier ring in said opening and pipe slips suspended from said carrier ring, said supporting ring and said carrier ring each having a segment thereof detachable from the remainder of the ring, at least at one end of the segment, to provide a lateral pipe opening whereby said rings may be applied laterally to a pipe intermediate its ends.

14. A device according to claim 13, wherein said means for rotatably supporting said carrier ring comprises rollers journaled on radially extending axes on said supporting ring and a peripheral groove in said carrier ring providing upper and lower raceways between which said rollers are confined.

15. A device according to claim 13 or 14, wherein said carrier ring segment is detachable at both ends thereof from its mating segment, and wherein means are provided for temporarily securing each of the carrier ring segments to the corresponding supporting ring segments.

16. A device according to any of the preceding claims, wherein said slip-supporting member or arm is provided with a sectional supporting ring comprising pivotally interconnected sections, a segmental slip-carrier ring supported by said supporting ring and comprising detachable segments and means for releasably securing each carrier ring segment to a corresponding supporting ring section.

17. A device according to claim 16, wherein one of said securing means comprises a locking member mounted on a supporting ring section and movable into and out of interlocking engagement with a slip-carrier ring segment.

18. A device according to claim 16 or 17, wherein one of said securing means comprises a substantially U-shaped clamp member removably supported by a supporting ring section and having supporting engagement with a corresponding slip-carrier ring segment.

19. A device according to any of the preceding claims, including a double-acting piston and cylinder assembly connected to said slip-supporting member or arm for moving said member or arm upwardly and downwardly to release and set said slips, a pair of fluid supply conduits communicating with said

cylinder means at opposite sides of said piston means and adapted to be alternately connected with a source of pressure fluid, and latch means comprising cooperating latch members, one movable vertically with said member or arm and the other fixed against vertical movement, said latch members being operable, when engaged, to prevent descent of said member or arm and setting of said slips, means yieldingly urging one of said latch members toward engaging position and fluid-actuated means connected with both of said fluid supply conduits and with said last-named latch member for rendering the latter inoperative while pressure fluid is supplied to either of said supply conduits.

20. A device according to claim 19, wherein said latch means comprises a first abutment member movable vertically with said slip-supporting member or arm, and a second abutment member fixed against vertical movement and movable into and out of the path of vertical movement of said first abutment member and yieldingly urged into said path, said fluid-actuated means being associated with said second abutment means and operable in response to fluid pressure in either of said fluid supply conduits for normally holding said second abutment member out of the path of movement of said first abutment member.

21. A device according to claim 19, wherein said first abutment member comprises a vertically extending rod having a downwardly facing shoulder thereon, and wherein said second abutment member comprises a body member having an opening extending vertically therethrough through which said rod and shoulder may pass freely in one position of said body member, said body member being movable by said resilient means into a position where it is engaged by said shoulder to prevent downward movement of the slip-supporting member or arm when the supply of pressure fluid is cut off from both of said supply conduits.

22. A device for actuating well pipe slips, including a slip-supporting arm supported adjacent one end thereof on vertically reciprocating actuating means for reciprocation therewith, the other end of said arm being shaped to provide a supporting ring defining a circular opening, said actuating means being adapted to be mounted adjacent a bowl member having a downwardly and inwardly tapered slip seat, and said arm extending substantially horizontally from said actuating means to dispose said supporting ring above and concentric with said slip seat, a slip carrier ring rotatably

615,015

11

supported in the opening in said supporting ring and having an unrestricted opening therein concentric with and at least as large as the opening through said bowl member, a plurality of links pivotally attached at their upper ends to said carrier ring at points spaced radially outwardly of the opening in said ring, and a pipe slip pivotally attached to the lower end of each of said links, said slips being tapered to conform to the taper of the seat in said bowl member, the length of stroke of said actuating means being sufficient to lift said slips entirely above the upper end of said slip seat, and the points of pivotal connection of opposite ends of said links with said slips and said carrier ring being so related as to cause said slips, when they are raised, to move radially outwardly into positions entirely outside the opening in said carrier ring.

23. A device according to claim 22, including means providing a permanent pivotal connection between one end of each link and its respective slip and means providing a quickly detachable pivotal connection between the other end of each link and said carrier ring.

24. A device according to claim 23, wherein each quick-detachable connection comprises a pair of spaced pivot lugs on said ring having aligned bores and a radially extending slot in each lug, said slot being of less width than the diameter of the lug bore, a pivot pin integral with the link and having a major transverse dimension fitting said lug bore and a minor transverse dimension less than the width of said slot, whereby the pivot pin may be inserted into and removed from the lug bore through said slot.

25. A device for actuating well pipe slips, having its parts constructed or adapted to operate substantially as herein described with reference to any of the embodiments illustrated in the accompanying drawings.

Dated the 14th day of August, 1946.

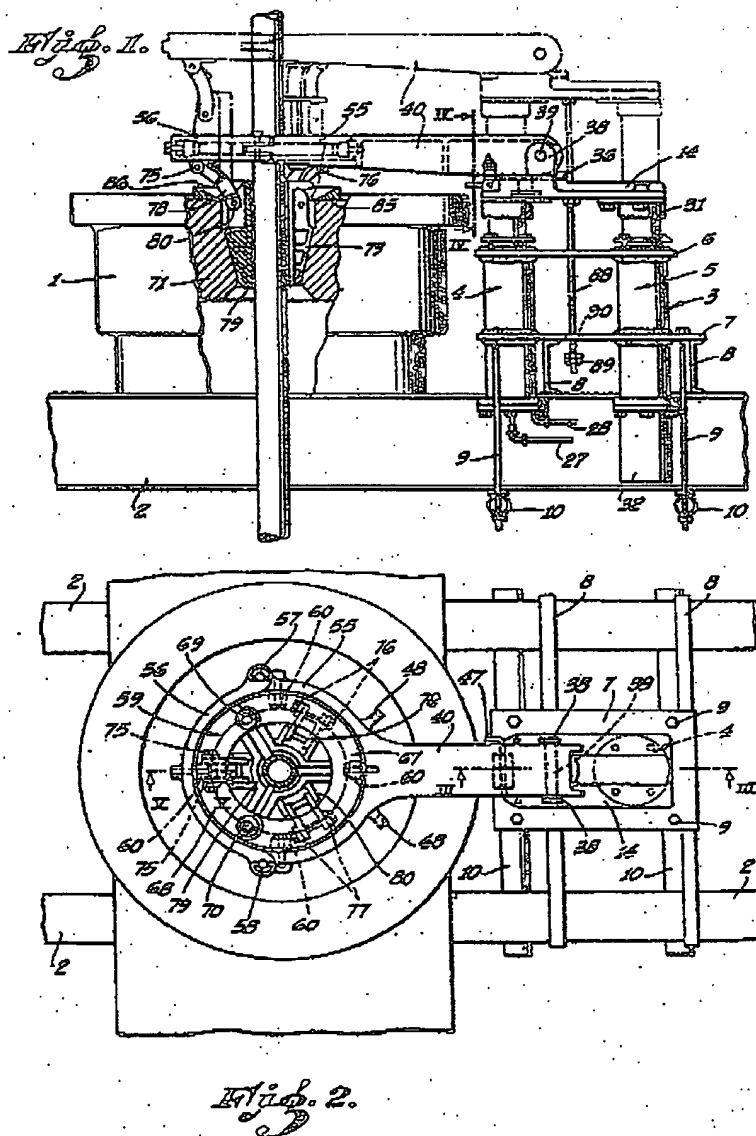
For: BYRON JACKSON CO.,
Stevens, Langner, Parry & Rollinson,
Chartered Patent Agents,
5/9, Quality Court, Chancery Lane,
London, W.C.2, and at
120, East 41st St., New York, U.S.A.

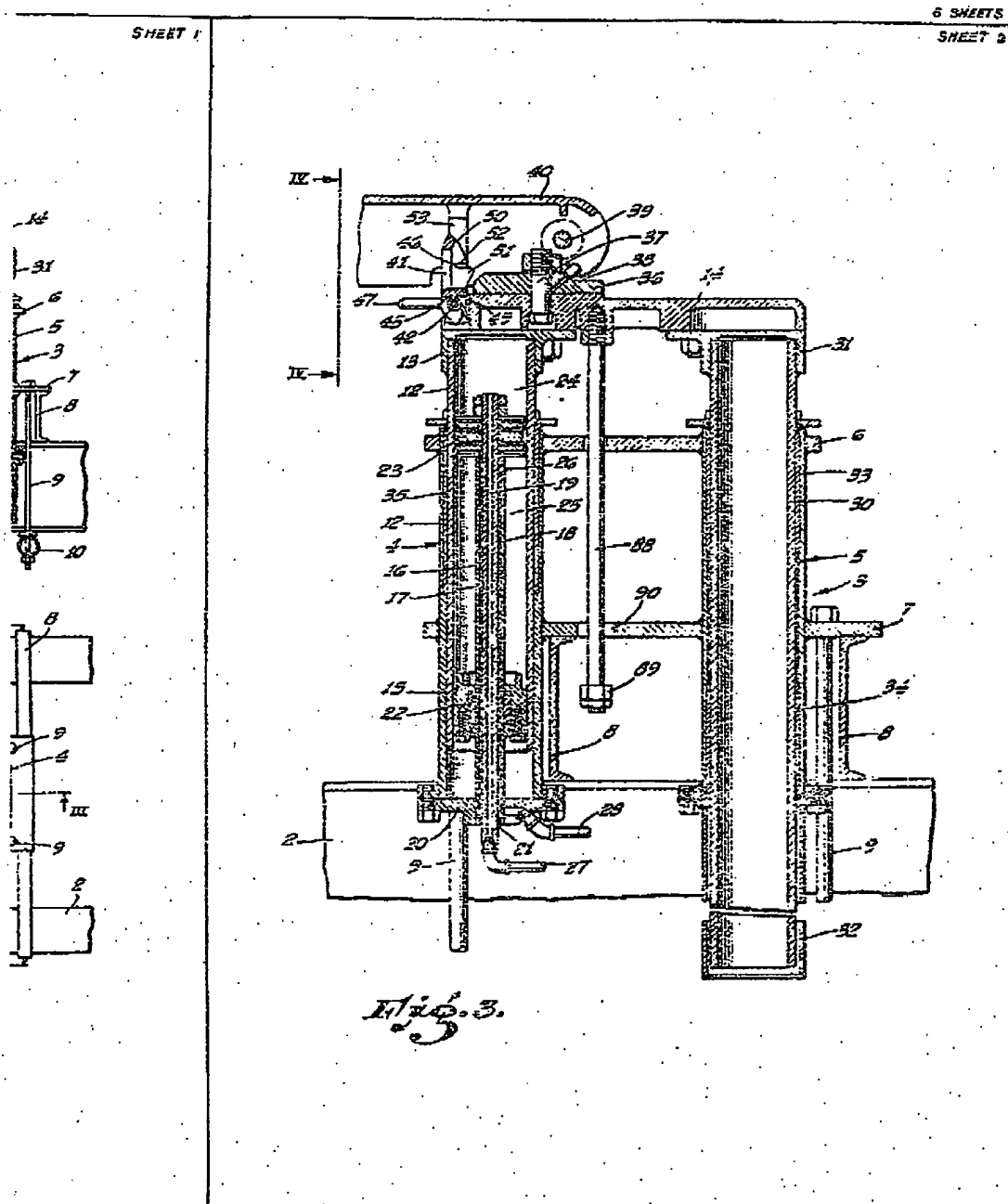
Leamington Spa: Printed for His Majesty's Stationary Office, by the Courier Press.—1949.
Published at The Patent Office, 25, Southampton Buildings, London, W.C.2, from which
copies, price 2s. 0d. each (inland) 2s. 1d. (abroad) may be obtained.

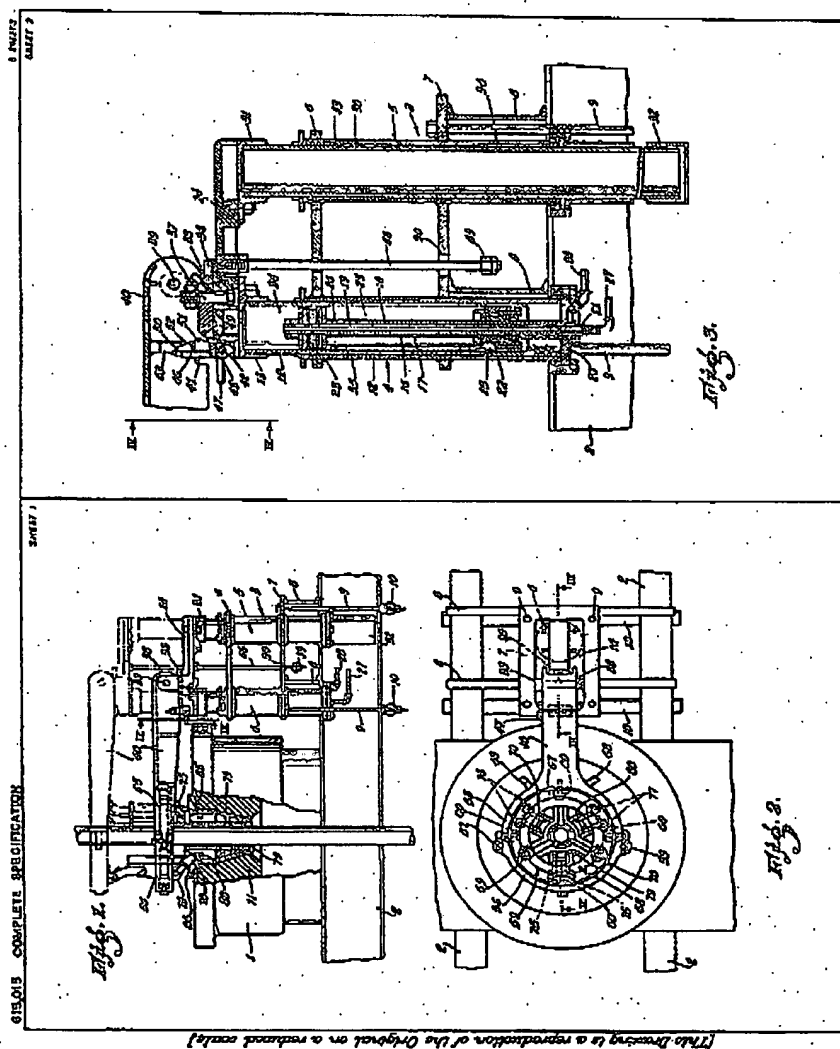
615,015 COMPLETE SPECIFICATION

5.

[This Drawing is a reproduction of the Original on a reduced scale.]







615,015 COMPLETE SPECIFICATION

SHE

[This Drawing is a reproduction of the Original on a reduced scale.]

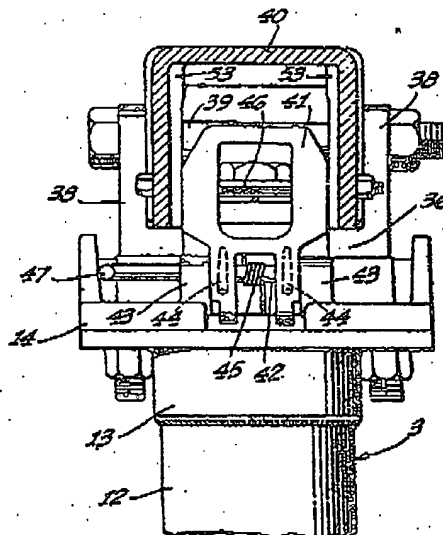


Fig. 4.

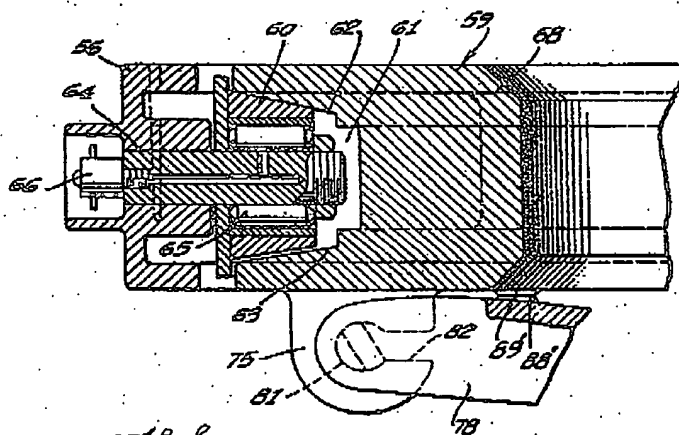


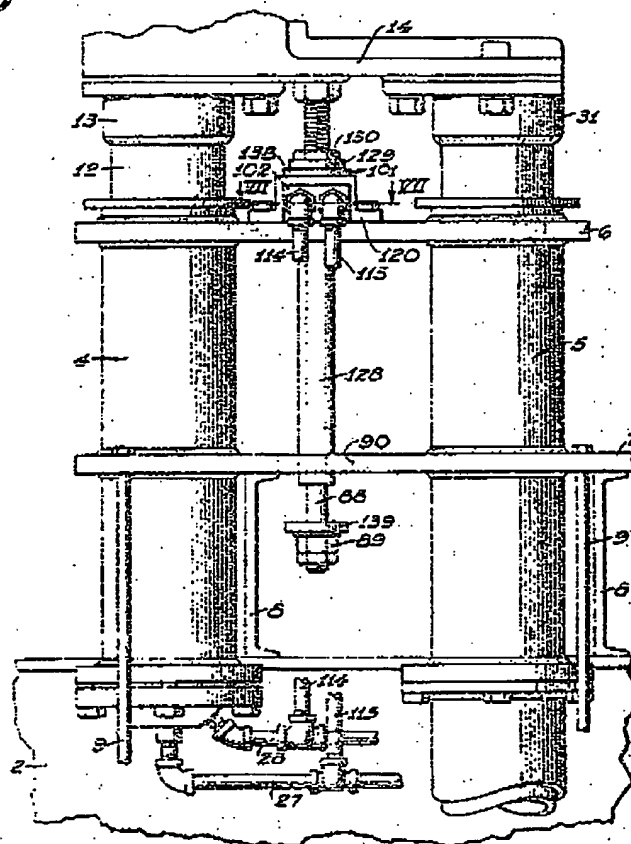
Fig. 5.

SHEET 3

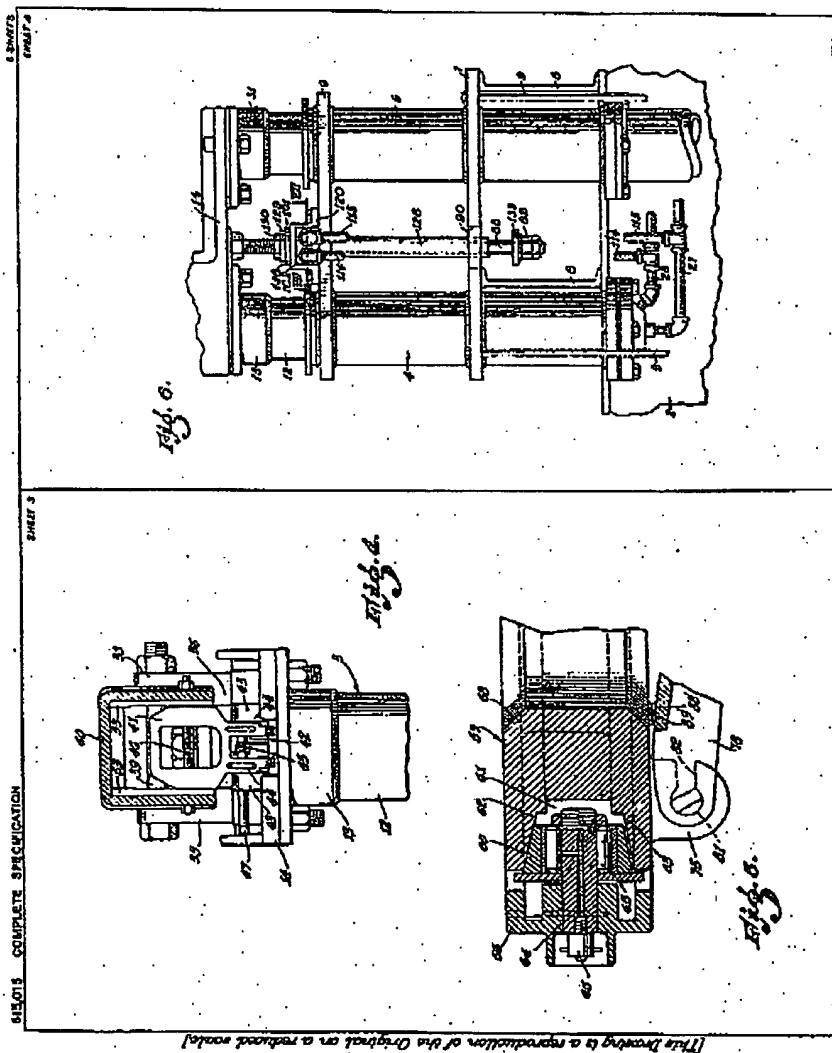
6 SHEETS

SHEET 4

Fig. 6.



H.M.S. 27. P.



615,015 COMPLETE SPECIFICATION

6 SHEETS
SHEET 5

Fig. 7.

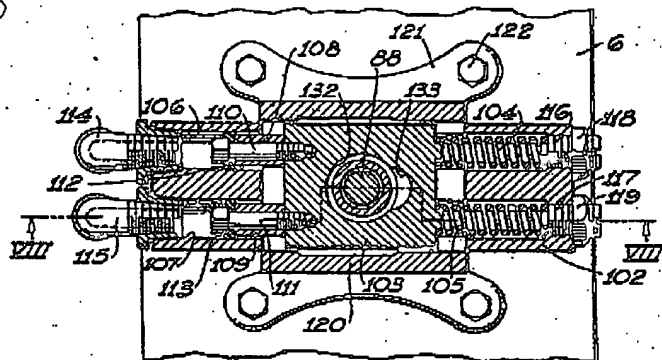
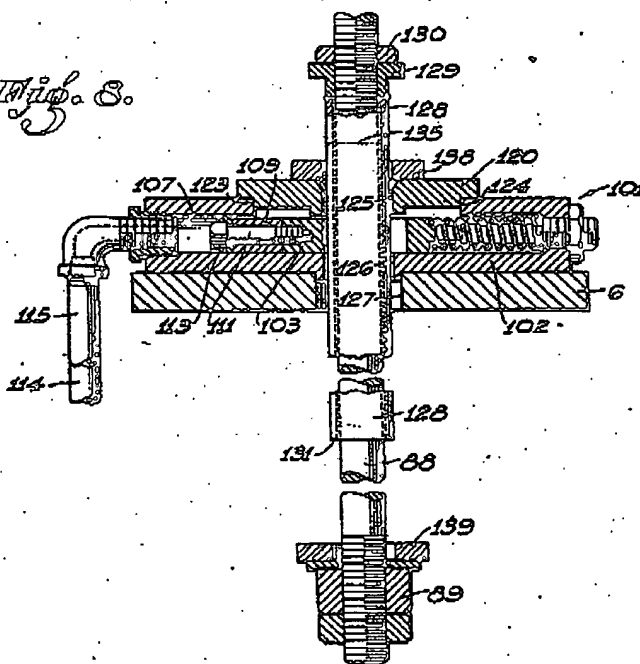


Fig. 8.

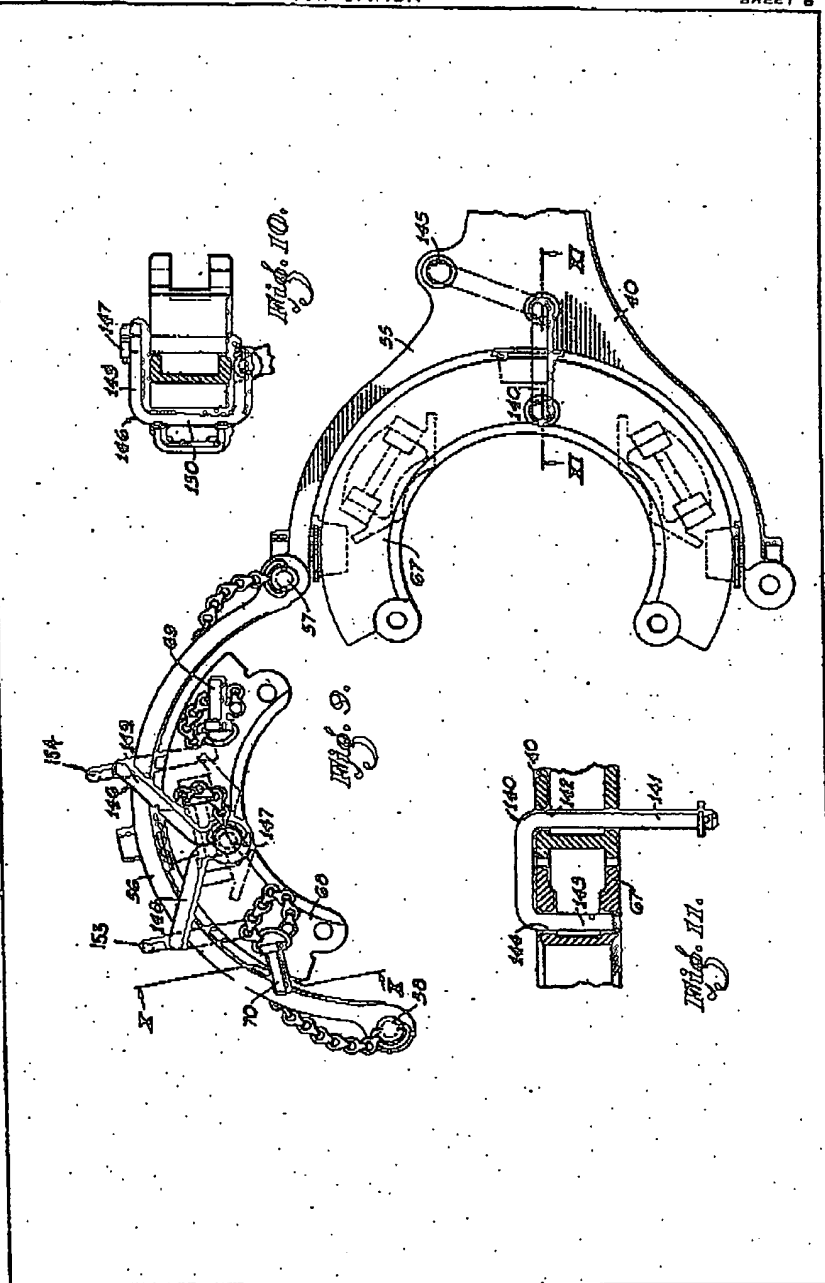


H.M.S.O. (By P.)

615,015 COMPLETE SPECIFICATION

6 SHEETS
SHEET 6

[This Drawing is a reproduction of the Original on a reduced scale.]



H.M.S.O. (71/2)

**This Page is Inserted by IFW Indexing and Scanning
Operations and is not part of the Official Record**

BEST AVAILABLE IMAGES

Defective images within this document are accurate representations of the original documents submitted by the applicant.

Defects in the images include but are not limited to the items checked:

- ☐ BLACK BORDERS
- ☐ IMAGE CUT OFF AT TOP, BOTTOM OR SIDES
- ☒ FADED TEXT OR DRAWING
- ☐ BLURRED OR ILLEGIBLE TEXT OR DRAWING
- ☐ SKEWED/SLANTED IMAGES
- ☐ COLOR OR BLACK AND WHITE PHOTOGRAPHS
- ☐ GRAY SCALE DOCUMENTS
- ☒ LINES OR MARKS ON ORIGINAL DOCUMENT
- ☐ REFERENCE(S) OR EXHIBIT(S) SUBMITTED ARE POOR QUALITY
- ☐ OTHER: _____

IMAGES ARE BEST AVAILABLE COPY.

As rescanning these documents will not correct the image problems checked, please do not report these problems to the IFW Image Problem Mailbox.